International Journal of Mechanical and Production Engineering Research and Development (IJMPERD) ISSN(P): 2249-6890; ISSN(E): 2249-8001 Vol. 6, Issue 3, Jun 2016, 23-34 © TJPRC Pvt. Ltd



## MODELING AND FINITE ELEMENT ANALYSIS OF

### VERTICAL AXIS WIND TURBINE ROTOR CONFIGURATIONS

# ASHWANI SHARMA<sup>1</sup> & M. A. MURTAZA<sup>2</sup>

<sup>1</sup>Assistant Professor, ASET, Amity University, Lucknow Campus, Uttar Pradesh, India <sup>2</sup>Professor ASET, Amity University, Lucknow Campus, Uttar Pradesh, India

#### ABSTRACT

This paper deals with the modeling and analysis of vertical axis wind turbine rotors (Savinious rotor; H rotor and their hybrid). Analysis includes stress and deformation distribution on rotors of different materials (Steel alloy, Aluminium alloy and Carbon Fibre Composite). Stress and deformation depend on the type of rotor and density of material.

KEYWORDS: Vertical Axis Wind Turbine, Mechanical Stress, Geometric Modeling, FEM, Rotor Configurations

Received: Apr 28, 2016; Accepted: May 12, 2016; Published: May 24, 2016; Paper Id.: IJMPERDJUN20163

#### INTRODUCTION

The wind turbine industry has experienced a huge development in the last 20 years. Wind turbine sizes have grown since then from 0.5 kW to 4.5MW. This growth is possible because of some scientific and product development strategies that are assisted by powerful computers and new software-tools.

The paper aims to do the comparative stress and deformation analysis of various Vertical Axis Wind Rotors and their hybrid configurations using Finite Element Analysis. In this work the modeling is done using Pro/E and the FEM analysis is done on Ansys.

#### VERTICAL AXIS WIND TURBINE

Considering the rotor axis position wind turbine can be classified as: Vertical axis wind turbine (VAWT) and horizontal axis wind turbine(HAWT).

Vertical Axis Wind Rotor [1] is known as cross wind axis machine. Here the axis of rotation is perpendicular to the direction of the wind and to the ground.

The main advantages of the vertical wind turbine are

- It accepts the wind from any direction
- Permits mounting of generator and gear box at the ground level
- VAWT require small working space so it can be mounted on the house roofs.
- Following are the disadvantages of a vertical wind turbine;
- Requires guy wires attached to the top for support, which may limit its application, particularly for

www.tjprc.org editor@tjprc.org

24

offshore sites

• Has lesser efficiency value.

Based on the applications the following are the commonly used;

- Savonious Rotor
- H rotor
- Darrius Rotor

This paper deals with the analysis of Savonious rotor and H rotor.

### HYBRID VERTICAL AXIS WIND ROTORS

A hybrid wind Rotor [2, 3] refers to the combination of two rotors on a single shaft. A combined rotor has better efficiency and high starting torque as compared to individual rotor.

Present study is for the hybrid configurations of Savonious-H rotor.

#### LITERATURE REVIEW

The vertical axis wind rotors Savonious [1] and Darrius [1] are often studied by many researchers. Most of them worked for improving the performance of the wind rotor, experimentally. Whereas the other factors such as stress and deformation produced in the material due to different loads acting on the wind rotor is not given much attention. The finite element analysis is mainly done for horizontal axis wind turbine by some manufacturers and few researchers. Performance improvement [3,4] has been attempted for hybrid rotors. R. Guptaet al. [2] studied the performance of Savonious rotor as well as Savonious-Darrius machine, experimentally. Two types of models, one Savonious rotor and other Savonious-Darrius machine were designed and fabricated and tested in subsonic wind tunnel. It was observed that there was improvement in the power coefficient for Savonious-Darrius machine compared to Savonious rotor under similar test conditions. Similarly, experimental work on Darrius rotor combined with the Savonious model was undertaken by Ali [6] in which theoretical results of the combined machine have been compared with the experimental results. The comparison shows a very good agreement between the two. As far as known no work is available for stress and deformation analysis of hybrid rotors.

In the present work, a comparative analysis of Savonious rotor, H- rotor and the combination of two has been done.

# MATHEMATICAL ANALYSIS OF VAWT

• Power Available in the Wind[5]

$$P = 0.5 \rho \text{ CpV}^3 \text{A} \eta_m \eta_e$$

P= power Available in the wind  $\rho$  = **density of air** 

**n**<sub>s</sub> = Electrical efficiency

*η*<sub>m</sub> =mechanical efficiency

V= wind speed

A= Swept area of rotor

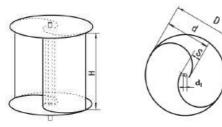
 $C_p$  = Coefficient of performance

## • Swept Area of the Rotor [5]

Savonious rotor

The area of Savonious Rotoris given by

A = H.D



**Figure 1: Savonious Rotor** 

Where,

H = height of Savonious rotor; d = diameter of cup

 $D = diameter of Savonious profile; d_1= shaft diameter; S is the overlap.$ 

• H Rotor

The area of H-Rotor [5] is given by

A=DH

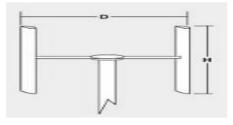


Figure 2: H Rotor

## • Torque Calculation

T = F \* R

T is the torque produced due to the force of the wind

F is the force acting; R is Radius of rotor

Also,

The torque acting can be calculated as

<u>www.tjprc.org</u> editor@tjprc.org

$$T = \frac{P}{\omega}$$

where, P is the output power and  $\omega$  is the angular velocity is given as follows.

$$\omega = \frac{2\pi n}{60}$$

Where n is the rotational speed in rpm.

By substituting these values, we get,

$$T = 9.55 \cdot \frac{P}{n}_{N-m}$$

Where P is in kilowatts (kW)

i.e. 
$$T = 9.55 \times 10^6 \frac{P}{n}$$
 N-mm; Where P is in kilowatts (kW)

### • Calculation of Rotational Speed

$$V = R\omega \quad \text{And} \quad \omega = \frac{2\pi n}{60}$$

From here we get

$$\mathbf{n} = \frac{60}{\pi} \frac{\mathbf{V}}{2\mathbf{R}}$$

Where V is the peripheral velocity of the rotor and is equal to the velocity of the win

### • Calculation for Hybrid Rotor

Force acting on the blades and the torque produced are calculated with an assumption that the total power output (shaft) is contributed equally by both the rotors using all the above relations

## **ASSUMPTIONS**

The various assumptions for modeling and analysis are stated below:

- The power output of all the wind rotors are same (here for analysis power output is taken as 1.1 KW)
- The Speed of the wind is same for all the rotors (here 15 m/s for analysis)
- The coefficient of performance i.e. Cp is taken as 0.3
- Mechanical and electrical efficiency is taken as 0.90 and 0.95 respectively.
- The diameter of all rotors is constant and taken as 1 meter; only their height is changed to fulfill the requirement of calculating swept area.
- The torque acting on all the rotors is same and so is the rotational speed.
- The forces acting on the rotors due to wind are same but this force is acting on different shapes of wind rotor1s,

therefore the same wind force along with the gravitational force will produce different stress for different shapes and so as the deformation.

• The width thickness of the blade is takes as 3 mm.

The values of different parameters are taken on the basis of the work carried out by previous researchers.

## GEOMETRIC MODELING

The modeling of the different wind rotors is done according to the dimensions calculated

Actual power available in the wind is

$$P = 0.5 \rho V^3 A C_p \eta_m \eta_{\epsilon}$$

Here, P=1.1KW, Cp= 0.3, V=15 m/sec,  $\eta_m$ =0.95,

$$\eta_e$$
=0.90,  $\rho$ =1.2255 Kg/m<sup>3</sup>

Putting these values in the above equations

1100= 530.233 A

$$A=2.07 \text{ m}^3$$

From these, the dimensions of rotor is calculated, as following,

#### **For Savonious Rotor**

A = D.H

A= calculated swept area and D= 1 meter and H= height of rotor

Therefore

H = 2.07 meter i.e. 2070 mm

And the cup diameter is D/2 i.e. 500 mm since there is no overlap.

#### For H-Rotor

D = 1000 and H = 2070

But here the diameter D is composed of following

D=2(L+r)

Herer = 80 mm

Therefore L = 420

## For Hybrid Rotors

The calculation is done in the similar manner as that of the above and then the Rotors are combined one over other to get the hybrid structure.

www.tjprc.org editor@tjprc.org

The power in this case will be half of the total power

550= 530.233 A

 $A=1.037 \text{ m}^2$ 

From this the dimension of hybrid rotor can be calculated

Savonious Rotor

D = 1000 mm and H = 1037 mm

H rotor

D = 1000 mm and H = 1037

Length of the shaft is taken according to the height of the rotor. With these, dimension modeling of the different rotors is done and then it is sends to ANSYS software for Finite Element Analysis.

#### LOADS CALCULATION

There are two major loads that are taken into consideration for the analysis:

- Load due to gravity
- · Load of forces due to wind

The load due to gravity depends upon the mass of the structure and acts in downward direction. It can be easily be calculated from

where Fg is the weight of the body is the mass of the structure and g is the acceleration due to gravity.

The wind force is calculated as following:

• The Rotational Speed of the Rotor is given by;

$$n = \frac{60}{\pi} \frac{v}{2R}$$

Here V is the wind speed in mm/sec, and R is the rotor radius in mm

$$n = (60.15000) / (3.14*2*500)$$

n = 286.62

n = 287 RPM

• The Torque is given by;

$$T = 9.55 X 10^6 \frac{P}{D}$$

T = 36602.78 N-mm

## • The Force Acting on the Blades is given by;

F = T/R

F = 36602.78/(500)

F = 73.21 N

For hybrid rotors, since the power is half and all the other factors are same, such as rotor diameter, the force acting on hybrid rotors is

F = 73.21/2

= 36.60 N

### **MATERIALS USED**

Following three materials have been considered for this study;

### • Plain Carbon Low Alloy Steel

The material used for the analysis is Plain Carbon Low Alloy Steel [7] (A36 having composition of Fe- 98%, Mn-1.0 %, C-0.29%, Si-0.28)

### • Aluminum Alloy

Aluminum Alloy [7] 6061 with chemical composition of 95.85% Al, 1.0 % Mg, 0.6% Si, 0.3 % Cu, and 0.20% Cr

### Carbon Fiber

Carbon Fiber Composite Material (Unidirectional Fiber/Epoxy Matrix) is taken for the analysis.

## FINITE ELEMENT MODELLING

The following figure shows the various constraints acting on the Savonious rotor:

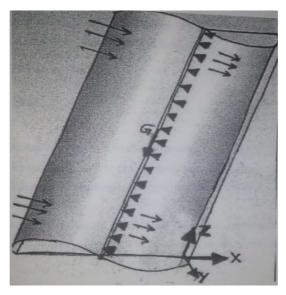


Figure 3: Boundary Conditions Acting on Savonious Rotor

www.tjprc.org editor@tjprc.org

 $U_x$ =Deformation in X Direction,  $U_y$ =Deformation in Y Direction,  $U_z$ =Deformation in Z Direction.

Wind Load in Y Direction, Gravity Load in Z Direction

# **DISCUSSION OF RESULTS**

Finite Element analysis has been done to obtain deformation and stress distribution in rotors.

#### Deformation

Figures 4, 5 & 6 show the deformation distribution in different rotors using stainless steel as material.

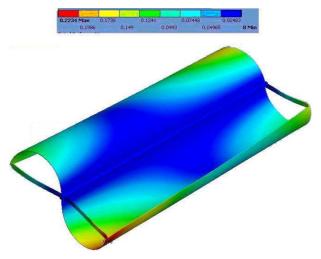


Figure 4: Deformation in Savonious Rotor

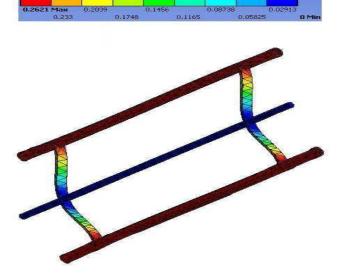


Figure 5: Deformation in H Rotor

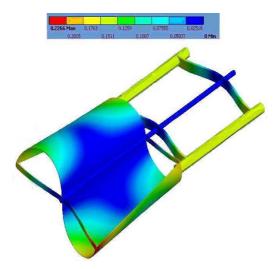


Figure 6: Deformation in Hybrid Rotor

## • Stress Distribution

The Equivalent stress acting on the model is plotted when stress display is required. The Von Mises stress for checking yield stress is chosen.

The stress generated in various individual and hybrid rotor is shown in figures 7, 8 & 9.

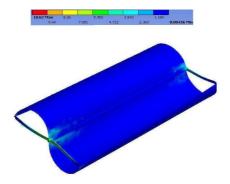


Figure 7: Stress in Savonious Rotor

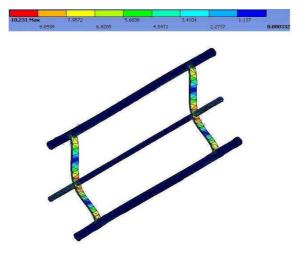


Figure 8: Stress in H Rotor

<u>www.tjprc.org</u> editor@tjprc.org

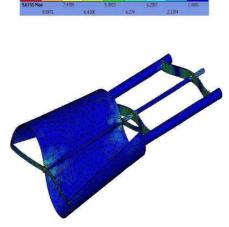


Figure 9: Stress in Hybrid Rotor

Similarly stress and deformation analysis was carried out using aluminum alloy and carbon fiber as material. The result of mass, deformation and stress for different material is tabulated (Tables 1, 2 & 3) and also depicted by graphically in figures 10, 11 & 12.

Mass(Kg) S. A1 Carbon Stee1 Alloy Fiber Savonious 146.74 52.447 30.294 H Rotor 120.23 42.971 24.821 Savonious 136.74 48.873 28.23 H rotor

**Table 1: Mass of Rotors** 

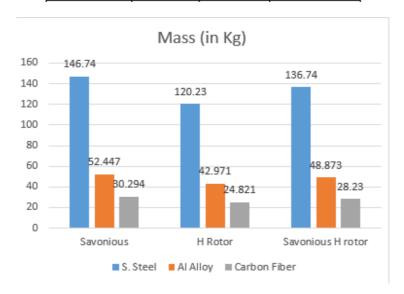


Figure 10: Mass of Rotors

**Table 2: Deformation in Various Rotors** 

	Deformation(mm)		
	S. Steel	Al alloy	Carbon Fiber
Savonious	0.2234	0.3006	0.1213
H Rotor	0.2621	0.3575	0.1556
Savonious H rotor	0.2266	0.2593	0.09365



**Figure 11: Deformation in Various Rotors** 

**Table 3: Stress in Various Rotors** 

	Stress (MPa)		
	S. Steel	Al alloy	Carbon Fiber
Savonious	10.62	5.268	4.046
H Rotor	10.231	3.9487	2.4976
Savonious H rotor	9.6155	4.1869	2.938

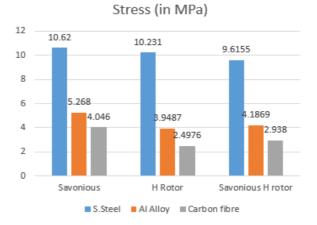


Figure 12: Stress in various rotor

# **CONCLUSIONS**

On the basis of the above modeling and analysis, following conclusions are made:

<u>www.tjprc.org</u> editor@tjprc.org

 Maximum deformation is produced in H Rotor and minimum in Savonious rotor, when the material used is Steel alloy.

When material used is Aluminium alloy, the maximum deformation is produced in H- Rotor and minimum in Savonious-H rotor.

Whereas, in the case of Carbon Fibre Composite, the maximum deformation is produced in H- Rotor, and the minimum in Savonious – H Rotor.

• The maximum stress is produced in Savonious Rotor and minimum in Savonious-H-Rotor, when the material used is Steel alloy.

In case of Aluminium alloy, The maximum stress is produced in Savonious Rotor and minimum in H Rotor.

Similarly when for Carbon Fiber Composite, the maximum stress is produced in Savonious Rotor and minimum in H Rotor.

One thing is worth noting, that both, the deformation and stress values varies drastically with the density of the material. The material with less density shows minimum deformation and stress.

Thus on the basis of this analysis if deformation is criteria then hybrid configuration of Savonious-H rotor made of Carbon fiber show least deformation.

On the other hand, when stress is design criteria H-rotor made of carbon fiber shows minimum induced stress.

The present work is based on the maximum value of stress and deformation as calculated by ANSYS software that generates an approximate solution. So some analytical or experimental method should be developed for calculating exact values of stress and deformation. Also, materials having less density and good strength can be tried for the same analysis.

#### REFERENCES

- 1. Rai G.D. (2006)," Non-Conventional Sources of energy", Khanna Publishers, Delhi pp 227-310
- 2. R. Gupta, R. Das & K.K. Sharma, (2006) "Experimental Study of Savonious Darrius Wind Machine" Proceedings of the International Conference on Renewable Energy for developing Countries-2006
- 3. Yusaku Kyozuka, Hiroyuki Akira, Di Duan, and Yuichiro Urakata "An Experimental Study on the Darrieus-Savonius Turbine for the Tidal Current Power Generation" (2009) Proceedings of the Nineteenth (2009) International Offshore and Polar Engineering Conference Osaka, Japan, June 21-26, 2009 Page 349
- 4. William S. Becker, (November 7, 2006), "Wind Turbine Device" United States Patent no. US 7,132,760 B2.
- 5. Paul Gipe, (2004) "Wind power renewable energy for home farm and business", Chelsia Green Publication Company, White River Junction, VT, Page no. 61.
- 6. Ali Mohamed Elmabrok, (March, 2009), Estimation of "The Performance of the Darrius Savonious Combined Machine" Ecologic Vehicles Renewable Energies Monaco, March 26-29.
- 7. W.D. Calister Jr. (2007) "Material Science and Engineering An Introduction" Wiley Publishers" New Delhi." John Wiley & Sons, Inc., Appendix B, Properties of Selected Engineering Materials page A3-A30.